## Model Question Paper-1 with effect from 2022-23 (CBCS Scheme)

USN


# Fourth Semester B.E. Degree Examination Subject Title: Aero Engineering Thermodynamics 

TIME: 03 Hours

Note: 01. Answer any FIVE full questions, choosing at least ONE question from each MODULE. 02. Use of Thermodynamic Data Hand Book is permitted

| Module -1 |  |  | *Bloom's <br> Taxonomy Level | Marks |
| :---: | :---: | :---: | :---: | :---: |
| Q. 01 | a | Distinguish between i) Microscopic and macroscopic approaches of thermodynamics <br> ii) Intensive and extensive Properties <br> iii) Open system and Closed system <br> iv) Point function and path function | L2 | 10 |
|  | b | A constant volume gas thermometer containing helium gives reading of gas pressure $p$ of 1000 mm Hg and 1366 mm Hg at ice point and steam point respectively. Assuming a linear relationship of the form $t=\alpha+\beta p$, express the gas thermometer Celsius temperature $t$ in terms of gauge pressure p . What is the temperature recorded by the thermometer when it registers a pressure of 1074 mm Hg ? | L3 | 10 |
| OR |  |  |  |  |
| Q. 02 | a | A system undergoes a process in which the pressure and volume are related by an equation of the form $\mathrm{pv}^{\mathrm{n}}=\mathrm{a}$ constant. Derive an expression for displacement work during this process. | L2 | 6 |
|  | b | Gas confined in a cylinder by a piston at a pressure of $3 \times 105 \mathrm{~N} / \mathrm{m} 2$ and a volume of 12000 cm 3 changes its state in such a way that the pressure is inversely proportional to the volume in this process if the pressure Falls to $1 / 3$ of its initial value find the magnitude and direction of workflow | L3 | 8 |
|  | c | Derive an expression for work done for the following process <br> i. Constant Pressure process <br> ii. Constant Volume process <br> iii. Constant Temperature process | L2 | 6 |
| Module-2 |  |  |  |  |
| Q. 03 | a | Starting from the first law of thermodynamics for a closed system undergoing a non-cyclic process derive the steady state, steady flow energy equation for a control volume (open system) | L4 | 6 |
|  | b | Describe an expression for heat transfer per unit mass for a reversible polytrophic process taking place in a closed system containing an ideal gas | L2 | 6 |
|  | c | A fluid system undergoes a non-flow frictionless process following the pressure volume relation as $p=5 / v+1.5$ where $p$ in bar $v$ is in $m 3$ during the process volume changes from 0.15 to 0.05 m 3 and the system rejects 45 kj of heat determine change in internal energy and change in enthalpy | L2 | 8 |
| OR |  |  |  |  |


| Q. 04 | a | Modify the general steady flow energy equation (SFEE) for the following cases <br> i)Steam turbine with negligible potential energy change if the process is adiabatic <br> ii)Horizontal steam nozzle with negligible entrance velocity of steam, if the process is non-adiabatic <br> iii)Insulated horizontal throttle valve | L2 | 10 |
| :---: | :---: | :---: | :---: | :---: |
|  | b | A turbine operating under steady flow conditions receives 4500 kg of steam per hour steam enters the turbine at a velocity of $2800 \mathrm{~m} / \mathrm{min}$ and an elevation of 5.5 m and specific enthalpy of $2800 \mathrm{kj} / \mathrm{kg}$ Its leaves the turbine at a velocity of $5600 \mathrm{~m} / \mathrm{min}$ elevation of 1.5 m specific enthalpy of $2300 \mathrm{kj} / \mathrm{kg}$ heat losses from the turbine to the surroundings amount to $16000 \mathrm{kj} / \mathrm{h}$ determine the power output of the turbine | L3 | 10 |
| Module-3 |  |  |  |  |
| Q. 05 | b | State the Kelvin-Plank and Clausius statements of the Second Law of thermodynamics and show that the violation of the former results in the violation of the later. | L2 | 8 |
|  |  | With a simple block diagram represent PMM of II kind and explain why it is not possible | L2 | 4 |
|  | c | Two reversible engines A and B are connected in series engine A receives heat from a thermal reservoir at T 1 and rejects at the temperature T2 while engine $B$ receives heat energy at T2 and rejects to a thermal reservoir at T3. If both engines are equal efficient show that the intermediate temperature T 2 is the geometric mean of T 1 and T 3 what will be the condition for intermediate temperatureT2 when both the engines deliver equal power. | L2 | 8 |
|  | OR |  |  |  |
| Q. 06 | a | Represent schematically heat engine, heat pump and refrigerator Prove that a reversible engine is more efficient than an irreversible engine operating between the same temperature limits. Give their performance | L2 | 8 |
|  | b | Prove that $\mathrm{COP}_{\text {heatpump }}=1+\mathrm{COP}_{\text {refrigerator }}$ | L2 | 4 |
|  | c | Heat pump to be used to heat the house in winter and reverse to cool the house in summer. The interior temperature is to be maintained at $20^{\circ} \mathrm{C}$. heat transfer through the walls and roof is estimated to be $0.525 \mathrm{~kJ} / \mathrm{s}$ per degree temperature difference. Between the inside and outside if the outside temperature in winter is $5^{\circ} \mathrm{C}$. What is the minimum power required to drive the heat pump. If the power requirement is same as in the part a what is the maximum outside temperature for which the inside can be maintained at $20^{\circ} \mathrm{C}$. | L2 | 8 |
|  | Module-4 |  |  |  |
| Q. 07 | a | Explain P-T diagram for a pure substance with the help of a neat sketch and define i) Triple point ii) Critical Point | L2 | 6 |
|  | b | Define the following i) Pure substance i) Triple Point iii) Quality and iv) Subcooled liquid | L2 | 6 |
|  | c | A vessel having a capacity of 0.05 m 3 contains a mixture of saturated water and saturated steam at a temperature of 245 C . The mass of the liquid present is 10 kg . Find the following i) The pressure ii) The Mass iii) The specific volume iv) The specific enthalpy v) The Specific entropy | L2 | 8 |
|  |  | OR |  |  |
| Q. 08 | a | Define dryness fraction of steam. What are the methods to measure dryness fraction. With a neat sketch explain any one method. | L3 | 6 |
|  | b | Derive an expression for TdS relation | L3 | 6 |
|  | c | Using steam tables, determine the mean specific heat for superheated steam | L3 | 8 |


|  |  | i)At 0.75 bar, between $100^{\circ} \mathrm{C}$ and $150^{\circ} \mathrm{C}$ <br> ii)at 0.5 bar between $300^{\circ} \mathrm{C}$ and $400^{\circ} \mathrm{C}$ |  |  |
| :--- | :--- | :--- | :--- | :--- |
| Module-5 |  |  | 10 |  |
| Q. 09 | a | With the help of P-V and T-S diagram, derive an expression for air <br> standard efficiency of Diesel cycle | L 3 | 10 |
|  | The pressure and temperature at the beginning of compression in an air <br> standard Otto cycle are 102 kPa and 315K. Heat is added during the <br> process at the rate of $250 \mathrm{~kJ} / \mathrm{kg}$ of air and air is used with a compression <br> ratio of 9. Assuming $\gamma=1.4$ and R = $287 \mathrm{~J} / \mathrm{kgK}$ for air, determine i)The <br> thermal efficiency of the cycle ii) The maximum Cycle temperature iii) <br> The maximum cycle pressure iv) Mean effective pressure | $\mathrm{L3}$ | 10 |  |
| Q. 10 | a | With the help of neat schematic diagram explain the working of Rankine <br> cycle and derive an expression for efficiency. | $\mathrm{L3}$ | 10 |
|  | b | In a steam power cycle, the steam supply is at 15 bar and dry and <br> saturated. The condenser pressure is 0.4 bar, Calculate the Carnot and <br> Rankine efficiencies of the cycle. Neglect pump work. | L 3 | 10 |

*Bloom's Taxonomy Level: Indicate as L1, L2, L3, L4, etc. It is also desirable to indicate the COs and POs to be $^{\text {a }}$ attained by every bit of questions.


# Fourth Semester B.E. Degree Examination AERO ENGINEERING THERMODYNAMICS 

TIME: 03 Hours
Max. Marks: 100
Note: 01. Answer any FIVE full questions, choosing at least ONE question from each MODULE. 02. Use of Thermodynamics tables are permitted.


|  | c | One Kg of water at 273 K is heated to 373 K by first bringing it in contact with reservoir at 323 K and then reservoir at 373 K . Determine the change in entropy of the universe? | L5 | 5 |
| :---: | :---: | :---: | :---: | :---: |
| Module-4 |  |  |  |  |
| Q. 07 | a | Define: (i) Compressibility factor (ii) Triple point (iii) Critical point (iv) Latent heat | L1 | 8 |
|  | b | Construct Vander Waal's constants in terms of critical properties. | L3 | 8 |
|  | c | Determine the specific volume, enthalpy, and internal energy of wet steam at 18 bar, dryness fraction 0.85 | L5 | 4 |
| OR |  |  |  |  |
| Q. 08 | a | Write Maxwell relation and explain the terms involved. | L2 | 4 |
|  | b | A rigid vessel of volume $0.3 \mathrm{~m}^{3}$ contains 10 Kg of air at 300 K . Determine the pressure that would be exerted by air on the vessel, using (i) Perfect gas equation (ii) Vander Waal's equation. Take for air, $\mathrm{R}=287.1 \mathrm{~J} / \mathrm{Kg} \mathrm{K}$, Molecular weight $=$ 28.96, Vander Waal's constants, $\mathrm{a}=135.8 \mathrm{kN} \mathrm{m}{ }^{4}(\mathrm{Kg} . \mathrm{mol})^{2}, \mathrm{~b}=0.0365$ $\mathrm{m}^{3} / \mathrm{Kg} . \mathrm{mol}$. | L5 | 8 |
|  | c | $0.1 \mathrm{~m}^{3}$ of air at $5 \mathrm{MPa}, 356^{\circ} \mathrm{C}$ contained in a cylinder expands reversibly and isothermally to 0.25 MPa . Calculate for air (i) Work transfer (ii) Heat transfer (iii) Change in entropy, assuming that air behaves as an ideal gas with $\mathrm{R}=287 \mathrm{~J} / \mathrm{Kg}$ K | L5 | 8 |
| Module-5 |  |  |  |  |
| Q. 09 | a | Construct an expression for air standard efficiency of an Otto cycle, representing all the process on a P-V and T-S diagram. | L3 | 10 |
|  | b | Compare Otto and Diesel cycles. | L4 | 6 |
|  | c | A Carnot engine rejects heat to the heat sink at $32^{\circ} \mathrm{C}$ and has a thermal efficiency of $52.3 \%$. The work output from the engine is 120 kJ . Determine: (i) The maximum working temperature of the engine and (ii) The heat added in kJ . | L5 | 4 |
| OR |  |  |  |  |
| Q. 10 | a | Sketch the schematic diagram and corresponding TS and HS diagram, derive an expression for efficiency of Rankine cycle. | L3 | 10 |
|  | b | In a single heater regenerative cycle, the steam enters the turbine at 30 bar, $400^{\circ} \mathrm{C}$ and the exhaust pressure is 0.10 bar. The feed water heater is a direct contact type which operates at 5 bar. Find (i) The efficiency of the cycle (ii) The increase in meant temperature of heat addition, efficiency, and steam rate, as compared to Rankine cycle (without regeneration). Neglect pump work. | L5 | 10 |

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